

Correspondence

On a Sub-Program to determine the Rayleigh Quotient by Conjugate Gradient Method

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Introduction

This paper is represented for a sub-program in order to determine the eigenvalue with respect to the continuous elastic system by applying the conjugate gradient method to Rayleigh quotient. This numerically computational method has been known as iterative method, and frequently used in order to compute the algebraic equation with several variables or large dimensional algebraic equation for elastic-plastic problems by Yamada [1]. The author also reported for eigenvalue problems of the continuous system in which this method were applied to Rayleigh quotient [2].

The advantages which this method is used are expressed in Ref. [3], therefore they are abbreviated in this paper. In applying this method Rayleigh quotient is used in order to numerically compute the eigenvalue instead of directly computing its from other method. The results are illustrated for the typical engineering examples.

1 Analysis System for Eigenvalue

Consider the continuous elastic system subjected to the concentrated axial forces, as shown in Fig. 1. In which ρ , A , EI and L are density per unit length, cross-sectional area, bending rigidity and total length of beam, respectively.

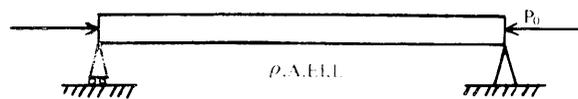


Fig. 1. Simply supported beam subjected to axial forces

The characteristic equations of eigenvalue of dynamic and static problems are generally expressed as,

$$\{[Ke] - P_0[Kg] - \Omega^2[M]\} \bar{q} = 0, \quad (1)$$

or

$$\{[Ke] - \lambda[Kg]\} = 0. \quad (2)$$

In which $[Ke]$, $[Kg]$ and $[M]$ are total stiffness, stability and mass matrices, respectively, Ω eigen circular frequency when the beam is subjected to the concentrated axial loads,

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λ critical value of buckling. Note that Eqs. (1) and (2) are formulated by the standard finite element method. The solution of the above equations can be obtained by computing the determinant as follow,

$$| [Ke] - P_0 [Kg] - \Omega^2 [M] | = 0, \quad (3)$$

or

$$| [Ke] - \lambda [Kg] | = 0. \quad (4)$$

Note that eigen circular frequency of free vibration can be directly obtained from Eq.(3) in $P_0=0$.

2 Algorithm of Conjugate Gradient Method

In this paper, Rayleigh quotient is used in order to obtained the eigenvalue instead of directly computing of Eqs. (3) and (4) stated above. The algorithm is shown as follow. The Rayleigh quotient for dynamic and static problems is expressed as,

$$R_i = \frac{\vec{q}_i^T [M] \vec{q}_i}{\vec{q}_i^T [K] \vec{q}_i}, \quad (5)$$

in which note that matrix $[Kg]$ in the stability problems is corresponding to matrix $[M]$ in the dynamic problems.

The value of this quotient will be a minimum when the displacement vector \vec{q} is that which most closely approximates the actual mode shape. The applying the conjugate gradient method to Eq. (5), it is, in the first, necessary to find the \vec{g}_i of R_i . Differentiating Eq. (5) with respect to \vec{q}_i^T in order to obtain \vec{g}_i yields subsequently,

$$\frac{\partial R_i}{\partial \vec{q}_i^T} = \frac{2}{\vec{q}_i^T [M] \vec{q}_i} ([K] \vec{q}_i - R_i [M] \vec{q}_i),$$

therefore, the direction of gradient is defined by

$$\vec{g}_i = [K] \vec{q}_i - R_i [M] \vec{q}_i. \quad (6)$$

The minimization proceeds step i to step $i+1$ by computing vector \vec{q}_{i+1} in step $i+1$ from vector \vec{q}_i in step i as follow,

$$\vec{q}_{i+1} = \vec{q}_i + \alpha_i \vec{P}_i.$$

where vector \vec{P}_{i+1} is orthogonal with respect to $[K]$, and α_i is determined by minimizing R_i . The substitution of Eq. (7) into Eq. (5) and its differentiation with respect to α_i yields subsequently,

$$\frac{\partial R_i}{\partial \alpha_i} = \frac{1}{\vec{q}_i^T [M] \vec{q}_i} (\vec{P}_i^T [M] \vec{q}_{i+1})(\vec{q}_i^T [M] \vec{q}_{i+1}) - (\vec{q}_{i+1}^T [K] \vec{q}_{i+1})(\vec{P}_i^T [M] \vec{q}_{i+1}). \quad (7)$$

The minimum value of R will occur where this partial derivative in zero. The extreme value will be obtained by solving the following equation,

$$(\vec{P}_i^T [K] \vec{q}_{i+1})(\vec{q}_i^T [M] \vec{q}_{i+1}) - (\vec{q}_{i+1}^T [K] \vec{q}_{i+1})(\vec{P}_i^T [M] \vec{q}_{i+1}) = 0. \quad (8)$$

On a Sub-Program to determine the Rayleigh Quotient by Conjugate Gradient Method

Substituting Eq. (7) into Eq. (8), the quadratic equation with respect to α_i can be subsequently obtained as,

$$a\alpha_i^2 + b\alpha_i + c = 0,$$

where the coefficients a , b and c are of the forms as,

$$\begin{aligned} a &= (\vec{P}_i^T [K] \vec{P}_i)(\vec{q}_i^T [K] \vec{P}_i) - (\vec{q}_i^T [K] \vec{P}_i)(\vec{P}_i^T [M] \vec{P}_i), \\ b &= (\vec{P}_i^T [K] \vec{P}_i)(\vec{q}_i^T [M] \vec{q}_i) - (\vec{q}_i^T [K] \vec{q}_i)(\vec{q}_i^T [M] \vec{P}_i), \\ c &= (\vec{q}_i^T [K] \vec{P}_i)(\vec{q}_i^T [M] \vec{q}_i) - (\vec{q}_i^T [K] \vec{q}_i)(\vec{P}_i^T [M] \vec{q}_i). \end{aligned}$$

The two roots with respect to α_i are computed from

$$\alpha_i = \frac{-b \pm \sqrt{b^2 - 4ac}}{2a}; \quad (i = 1, 2).$$

One of the two roots can be selected to minimizing quotient R obtained by substituting these into Eqs. (5) and (7), as shown in Fig. 2.

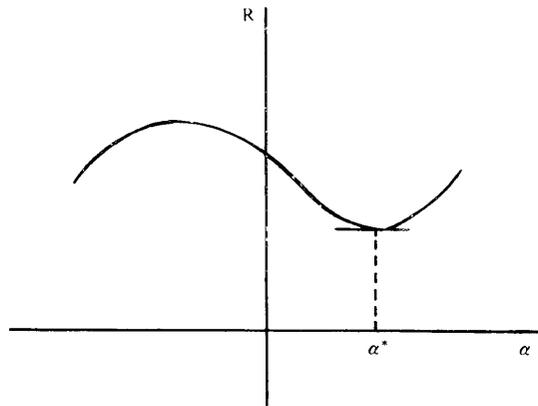


Fig. 2. Representative variation of Rayleigh quotient a line

Vector \vec{P}_{i+1} in step $i+1$ is determined from the gradient \vec{g}_{i+1} and vector \vec{P}_i obtained from the previous step by using the following formula,

$$\vec{P}_{i+1} = \vec{g}_{i+1} + \beta_i \vec{P}_i \quad (9)$$

where β_i is a constant coefficient and is found by requiring that \vec{P}_i and \vec{P}_{i+1} are orthogonal with respect to $[K]$. Therefore the value of β_i is given by,

$$\beta_i = -\frac{\vec{P}_i^T [K] \vec{g}_{i+1}}{\vec{P}_i^T [K] \vec{P}_i} \quad (10)$$

The algorithm can be summarized as,

1. Initialization ($i=0$)

$$\vec{q}_0 = \text{given arbitrary constant,}$$

$$\vec{g}_0 = R(\vec{q}_0),$$

$$\vec{P}_0 = \vec{g}_0.$$

2. Next step iteration

(1) compute α_i ,

(2) compute $\vec{q}_{i+1} = \vec{q}_i + \alpha_i \vec{P}_i$,

(3) compute $R_{i+1} = \frac{\vec{q}_{i+1}^T [K] \vec{q}_{i+1}}{\vec{q}_{i+1}^T [M] \vec{q}_{i+1}}$

(4) compute $\vec{g}_{i+1} = [K] \vec{q}_{i+1} - R_{i+1} [M] \vec{q}_{i+1}$

(5) compute $\beta = -\frac{\vec{P}_i^T [K] \vec{g}_{i+1}}{\vec{P}_i^T [K] \vec{P}_i}$

(6) compute $\vec{P}_{i+1} = \vec{g}_{i+1} + \beta \vec{P}_i$

The iteration are carried out to obtained the convergent value of R . The then convergent value can be given as follow,

$$|R_{i+1} - R_i| < e,$$

where e is the condition of coverage.

3 Numerical Example and Conclusion

Consider the simply supported uniform beam subjected to the concentrated axial forces at both ends, as shown in Fig. 1. In which mass, stiffness and stability matrices can be obtained by means of applying the standard finite element method, respectively. These are denoted as follow,

$$[m] = \frac{\rho A l}{420} \begin{bmatrix} 156 & 22l & 54 & -13l \\ 22l & 4l^2 & 13l & -3l^2 \\ 54 & 13l & 156 & -22l \\ -13l & -3l^2 & -22l & 4l^2 \end{bmatrix},$$

$$[k_e] = \frac{EI}{l^3} \begin{bmatrix} 12 & 6l & -12 & 6l \\ 6l & 4l^2 & -6l & 2l^2 \\ -12 & -6l & 12 & -6l \\ 6l & 2l^2 & -6l & 4l^2 \end{bmatrix},$$

$$[k_g] = \frac{P_0}{l} \begin{bmatrix} 6/5 & l/10 & -6/5 & l/10 \\ l/10 & 2l^2/15 & -l/10 & -l^2/30 \\ -6/5 & -l/10 & 6/5 & -l/10 \\ l/10 & -l^2/30 & -l/10 & 2l^2/15 \end{bmatrix}.$$

Each of the above element matrices must be assembled for the given entire structural system when these are employed to practical problems. R value corresponding to

On a Sub-Program to determine the Rayleigh Quotient by Conjugate Gradient Method

eigenvalue, ω^2 , of transverse free vibration and critical buckling value, P_{cr} , can be expressed as follow,

$$R = \omega^2 \frac{\rho AL^4}{EI} = \frac{\tilde{q}_i^T [Ke] \tilde{q}_i}{\tilde{q}_i^T [M] \tilde{q}_i} = \pi^4,$$

$$R = P_{cr} \frac{L^2}{EI} = \frac{\tilde{q}_i^T [Ke] \tilde{q}_i}{\tilde{q}_i^T [Kg] \tilde{q}_i} = \pi^2.$$

The results are shown for two kinds of the finite element in Table 1. The each of (2) and (4) in Table 1 expressed on the divided number of finite elements. The results are expressed that the four divided finite elements are sufficiently satisfied when the computation is carried out for the practical problems.

Table 1 A Comparison of eigenvalues

	$\sqrt{\omega^2 \frac{\rho AL^4}{EI}}$	error %	$P_{cr} \frac{L^2}{EI}$	error %
Theory	9.87		9.87	
Present solution	9.91(2)	0.4	9.94(2)	0.7
	9.88(4)	0.1	9.89(4)	0.2

The sub-program is shown in Appendix. Note that $EPS=10^{-5}$ for the condition of convergence and total length $L=450$ [mm] of the beam are given in order to carry out this computation.

4 Acknowledgement

The FACOM 230-38 in Kokushikan University Computer Center has been used for the objective computation.

(Recieved 18 January 1982)

References

- 1) Yamada, *et al.*, EPIC-IV, (in Japanese), (1980), BAIFUKAN.
- 2) K. Shimoyamada, *Prescript JSME*, (in Japanese), No. 760-1, (June, 1976), p 69.
- 3) K. Shimoyamada, *Memoris of the Kokushikan University Computer Center*, No.2, (march, 1981), p 15.
- 4) Rubinstein, M.F., *et al.*, *J. Franklin Inst.* Vol. 293, No. 3, (march, 1972), p173.

Appendix

This appendix shows the program for algorithm represented in this paper. When this program is used, we would be required to prepare the subroutine in order to operate, for example, the form of " $\tilde{q}^T [K] \tilde{q}$ " and to clear the vector (VECMAT and CLVEC in this program, respectively)

Memoirs of the Kokushikan University Computer Center No. 3

FACOM 230 CS2/V5 FORTHAN 5 V-03 L-70 DATE 82.01.28 TIME 14.26 PAGE 0001

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SOURCE LIST
C *** CONJUGATE GRADIENT METHOD *****
C AK-CORRESPONDING TO STIFFNESS MATRIX
C BM-CORRESPONDING TO MASS OR STABILITY MATRIX
C Q=DISPLACEMENT VECTOR
C D=JUDGEMENT OF ROOT OF QUADRATIC EQUATION
C NRED= DIMENSION FOR CALCULATION
C N=ELM=DIMENSION OF MATRIX
C SUBROUTINE FUMCG(AK,BM,RAL,D,NRED,NELM,Q,EPG)
0001 C **
0002 DIMENSION AK(NELM,NELM),BM(NELM,NELM),Q(NELM)
0003 DIMENSION P(30),Q0(30),QD(30),QLDP(30),OLDQ(30),C(30),G(30)
C **
C ** COMPUTE FOR INITIAL ARGUMENT*
0004 CALL CLVEC(Q,NELM)
C ARBITRARY VALUE GIVEN TO ALL COMPONENT OF DISPLACEMENT Q(I)
0005 DO 20 I=1,NRED
0006 Q(I)=1.0
C ** COMPUTE THE EACH FACTOR FOR INITIAL HAYLEIGH QUOTIENT
0007 CALL CLVEC(C,NELM)
0008 CALL VECMAT(C1,Q,Q,AK,C,NRED,NELM)
0009 CALL CLVEC(C,NELM)
0010 CALL VECMAT(C2,Q,Q,BM,C,NRED,NELM)
C COMPUTE THE INITIAL HAYLEIGH QUOTIENT
0011 HAL=C1/C2
0012 WRITE(6,50) RAL
C COMPUTE THE GRADIENT VECTOR GG(I)
0013 DO 60 I=1,NRED
0014 S=0.0
0015 DO 70 J=1,NRED
0016 S=S+(AK(I,J)-HAL*BM(I,J))*Q(J)
0017 70 CONTINUE
0018 GG(I)=S
0019 60 CONTINUE
C *** ORTHOGONAL VECTOR ***** P(I) ****
0020 DO 80 I=1,NRED
0021 P(I)=GG(I)
0022 80 CONTINUE
C *** NEXT STEP ITERATION *****
0023 K=0
0024 1000 CONTINUE
0025 K=K+1
0026 R=HAL
0027 DO 90 I=1,NRED
0028 OLDQ(I)=Q(I)
0029 QLDP(I)=P(I)
0030 90 CONTINUE
0031 CALL CLVEC(P,NELM)
0032 CALL CLVEC(Q,NELM)
0033 CALL CLVEC(GG,-NFM)
C COMPUTE THE FACTOR OF QUADRATIC EQUATION
0034 CALL VECMAT(D1,QLDP,QLDP,AK,C,NRED,NELM)
0035 CALL CLVEC(C,NELM)
0036 CALL VECMAT(D2,OLDQ,OLDQ,BM,C,NRED,NELM)
0037 CALL CLVEC(C,NELM)
0038 CALL VECMAT(D3,OLDQ,QLDP,AK,C,NRED,NELM)
0039 CALL CLVEC(C,NELM)
0040 CALL VECMAT(D4,QLDP,QLDP,BM,C,NRED,NELM)
0041 AA=(D1+D2)-(D3+D4)
0042 CALL CLVEC(C,NELM)
0043 CALL VECMAT(D1,QLDP,QLDP,AK,C,NRED,NELM)
0044 CALL CLVEC(C,NELM)
0045 CALL VECMAT(D2,OLDQ,OLDQ,BM,C,NRED,NELM)
0046 CALL CLVEC(C,NELM)
0047 CALL VECMAT(D3,OLDQ,QLDP,AK,C,NRED,NELM)
0048 CALL CLVEC(C,NELM)
0049 CALL VECMAT(D4,QLDP,QLDP,BM,C,NRED,NELM)
0050 CALL CLVEC(C,NELM)
0051 BB=(D1+D2)-(D3+D4)
0052 CALL VECMAT(D1,QLDP,OLDQ,AK,C,NRED,NELM)
0053 CALL CLVEC(C,NELM)
0054 CALL VECMAT(D2,OLDQ,OLDQ,BM,C,NRED,NELM)
0055 CALL CLVEC(C,NELM)
0056 CALL VECMAT(D3,OLDQ,OLDQ,AK,C,NRED,NELM)
0057 CALL CLVEC(C,NELM)
0058 CALL VECMAT(D4,QLDP,OLDQ,BM,C,NRED,NELM)
0059 CC=(D1+D2)-(D3+D4)
0060 CALL QUAD(AA,BB,CC,D,X1,X2)
0061 IF(D.LT.0.0) GO TO 1520
C *** COMPUTE THE DISPLACEMENT VECTOR QQ QD,*****
0062 DO 100 I=1,NRED
0063 QQ(I)=OLDQ(I)+X1*QLDP(I)
0064 QD(I)=OLDQ(I)+X2*QLDP(I)
0065 100 CONTINUE
C *** COMPUTE THE HAYLEIGH QUOTIENT HAL IN N-STEP
0066 HAL=0.0
0067 CALL CLVEC(C,NELM)
0068 CALL VECMAT(D1,Q0,Q0,AK,C,NRED,NELM)
0069 CALL CLVEC(C,NELM)
0070 CALL VECMAT(D2,Q0,Q0,BM,C,NRED,NELM)
0071 CALL CLVEC(C,NELM)
0072 CALL VECMAT(D3,QD,QD,AK,C,NRED,NELM)
0073 CALL CLVEC(C,NELM)
0074 CALL VECMAT(D4,QD,QD,BM,C,NRED,NELM)
0075 HAL1=D1/D2
0076 HAL2=D3/D4
0077 IF(HAL1.LT.HAL2.AND.HAL1.GT.0.0) GO TO 2000
0078 HAL=HAL2
0079 IF(HAL.LT.0.0) GO TO 1500

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